CHAPTER 15 HYDRAULIC PUMPS



Pump Family Tree

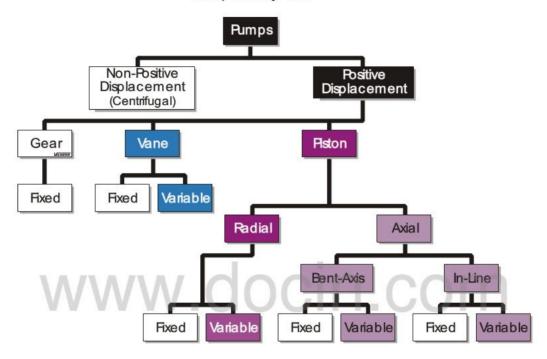


Figure 15-1 Pump categories



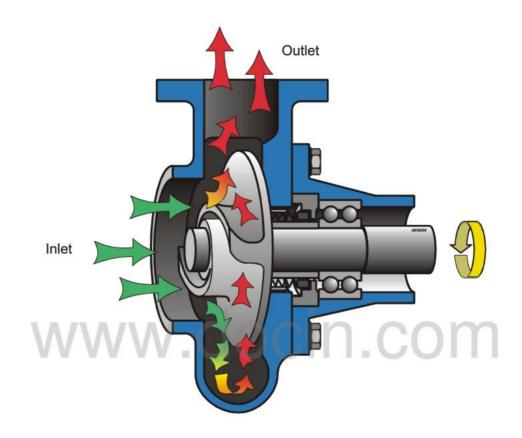


Figure 15-2 Non-Positive Displacement Pump



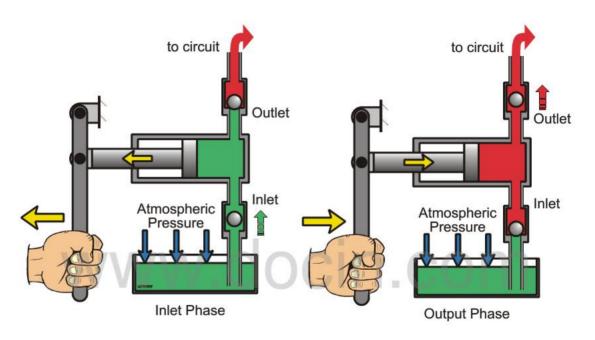


Figure 15-3 Positive displacement pump



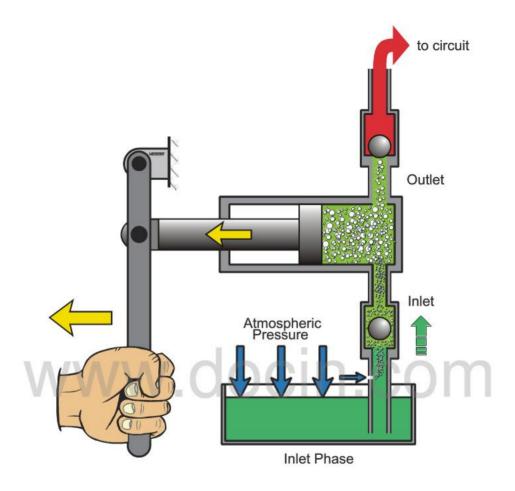


Figure 15-4 Aerated fluid caused by inlet air leak

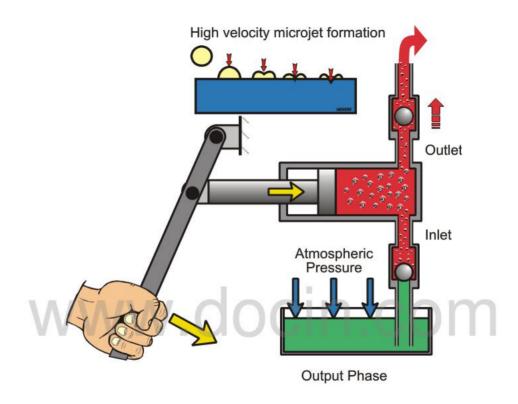


Figure 15-5 Pressurizing the fluid collapses the bubbles



Figure 15-6 Piston pump valve plate erosion caused by aeration or cavitation COPYRIGHT © (2001) EATON CORPORATION

FATCN

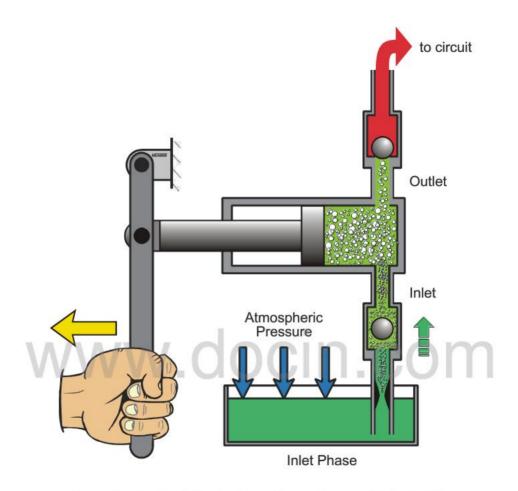


Figure 15-7 Cavitation bubbles formed by a restricted inlet COPYRIGHT © (2001) EATON CORPORATION



Causes of Cavitation and Aeration

Causes of Cavitation	Causes of Aeration
Clogged or restricted strainer	Low reservoir fluid level
2. High fluid viscosity	2. Defective pump shaft seal
3. Low fluid temperature	3. Return line above fluid level
4. Clogged reservoir breather	4. Improper baffling in the reservoir
5. Pump inlet line too small	5. Loose fitting on pump inlet line
6. Pump too far above reservoir	6. Defective seal on pump inlet line
7. Pump too far from reservoir	7. Incorrect reservoir design
8. Excessive pump RPM	8. Porous hose on pump inlet line
9. Too many bends in pump inlet line	ain aam
10. Collapsed hose on pump inlet line	CIII.COIII
11. Restriction on pump inlet line	
12. Failure of supercharge pump	

Figure 15-8 Causes of aeration and cavitation COPYRIGHT © (2001) EATON CORPORATION



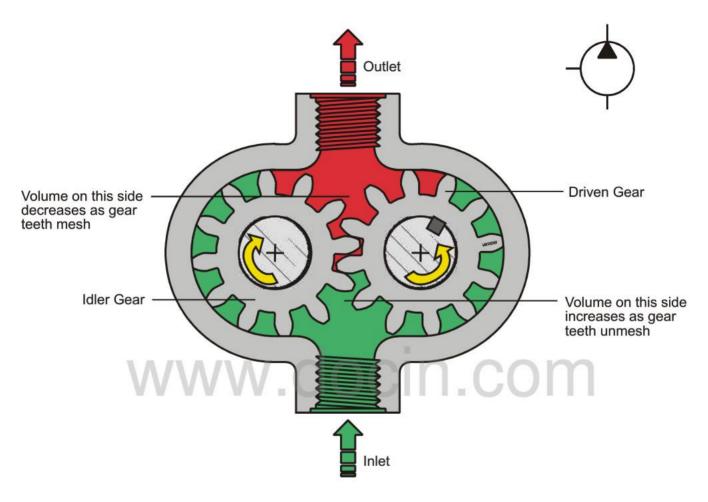


Figure 15-9 External gear pump

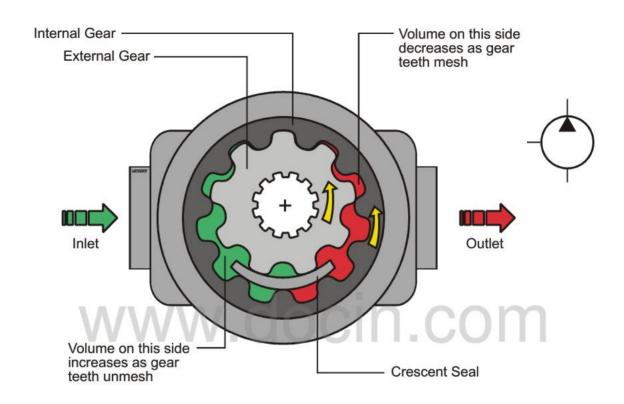


Figure 15-10 Crescent seal type internal gear pump

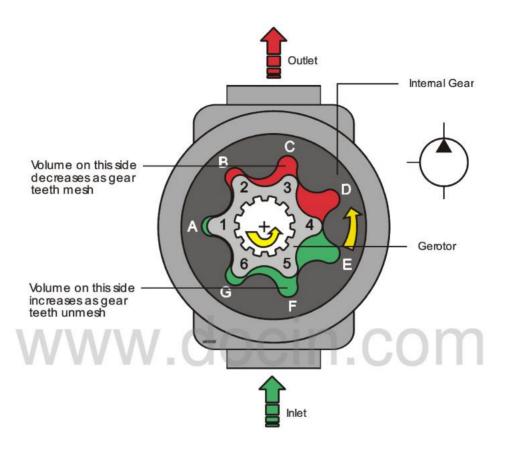


Figure 15-11 Gerotor Type Internal Gear Pump

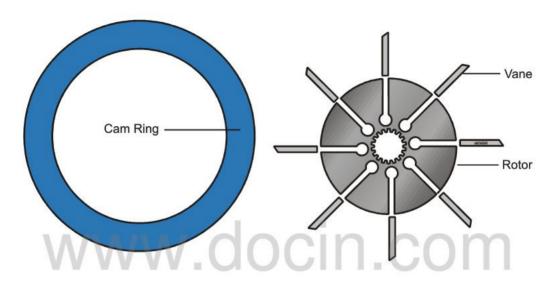


Figure 15-12 Basic pumping elements of a vane pump



Figure 15-13 Eccentricity between rotor and cam ring

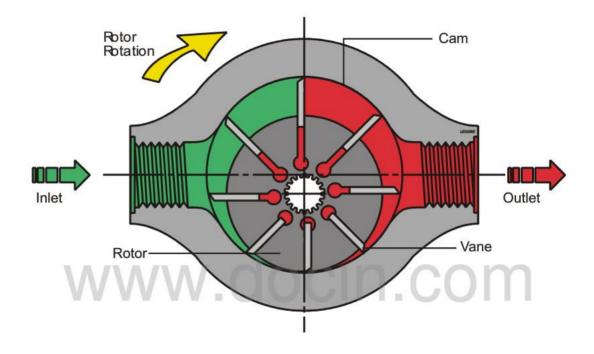


Figure 15-14 Rotating group with inlet and outlet ports

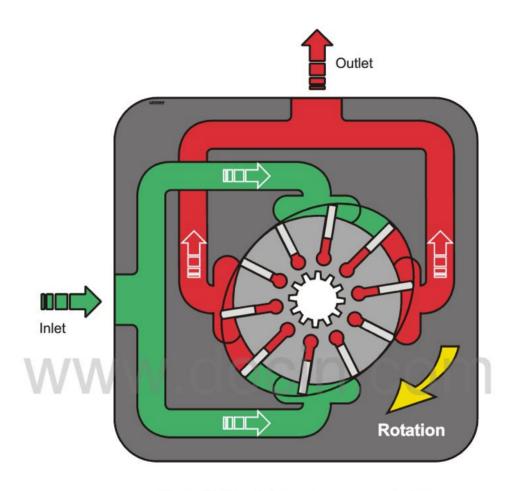


Figure 15-15 Balanced vane pump design

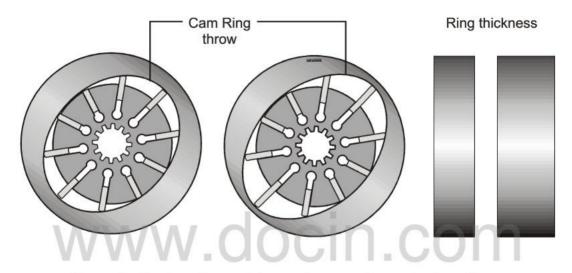


Figure 15-16 Interchangeable cam rings provide a selection of flows

COPYRIGHT © (2001) EATON CORPORATION

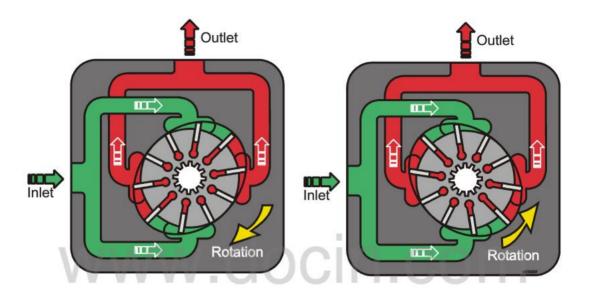


Figure 15-17 Rotating the cam ring 180° reverses the direction of rotation

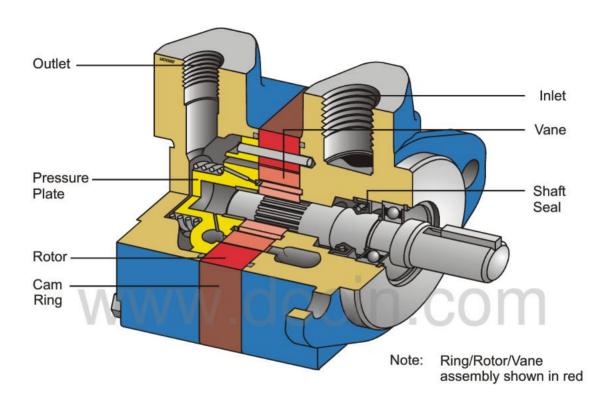


Figure 15-18 Fixed displacement 擦quare pump

Pressure balance across vane Vane Intra-vane Outlet Quadrant Reduced vane tip loading in the inlet quadrant

Figure 15-19 Vane and intra-vane assembly

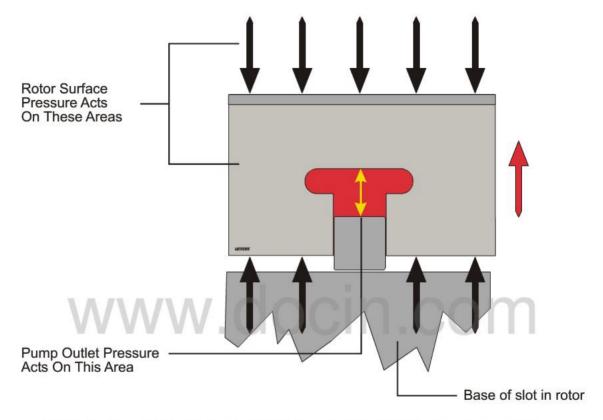


Figure 15-20 Outlet pressure in the intra-vane area assures vane-cam contact COPYRIGHT © (2001) EATON CORPORATION

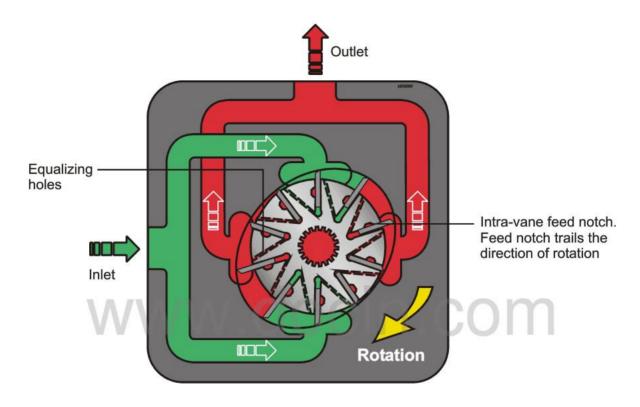


Figure 15-21 Porting in rotor and side plates provides vane pressure balance

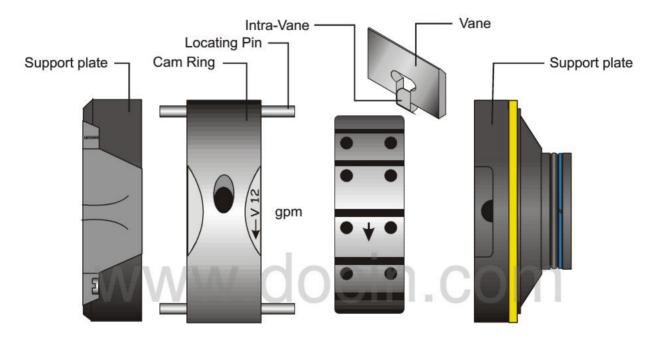


Figure 15-22 High performance pump cartridge components

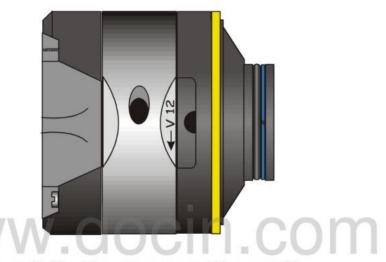


Figure 15-23 Intra-vane pump cartridge assembly



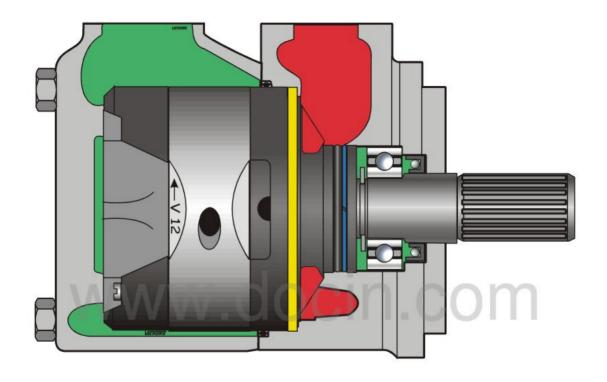


Figure 15-24 Intra-vane pump cartridge installed



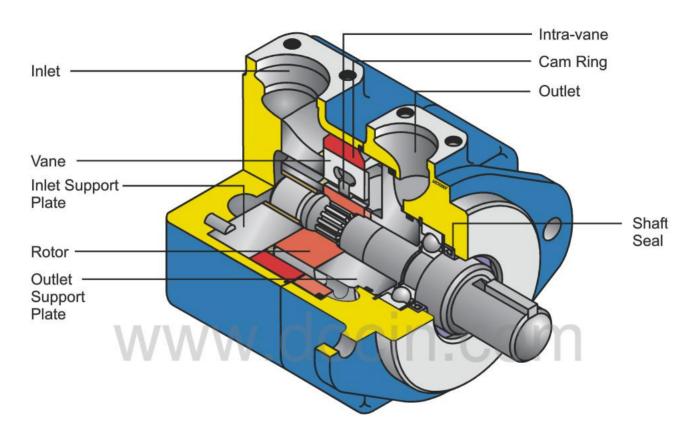


Figure 15-25 Intra-vane pump assembly

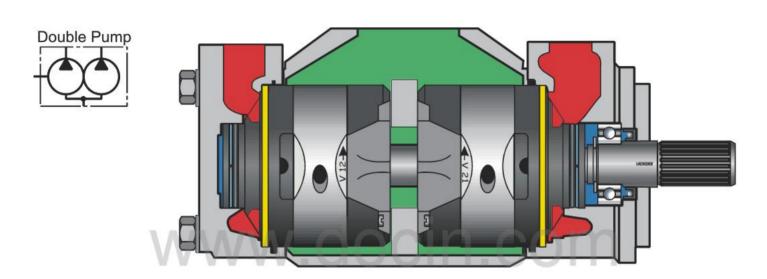


Figure 15-26 A double pump with a single inlet COPYRIGHT © (2001) EATON CORPORATION

FATON

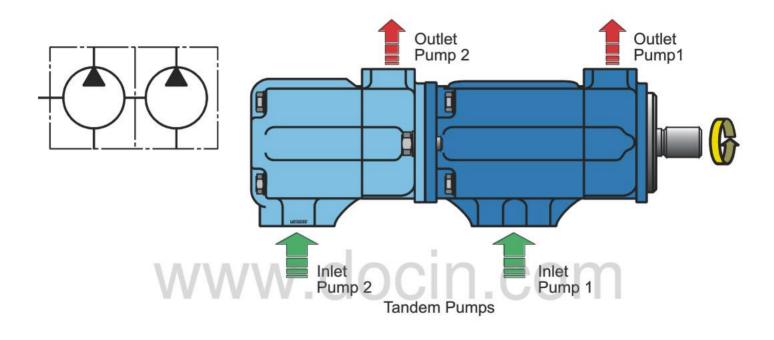


Figure 15-27 A double (tandem) pump with separate inlets



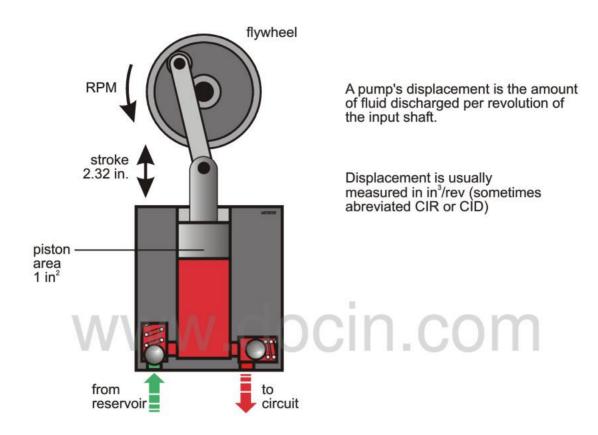


Figure 15-28 The piston pump concept

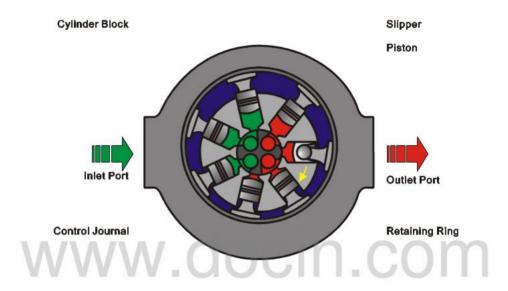


Figure 15.29 A Radial Piston Pump

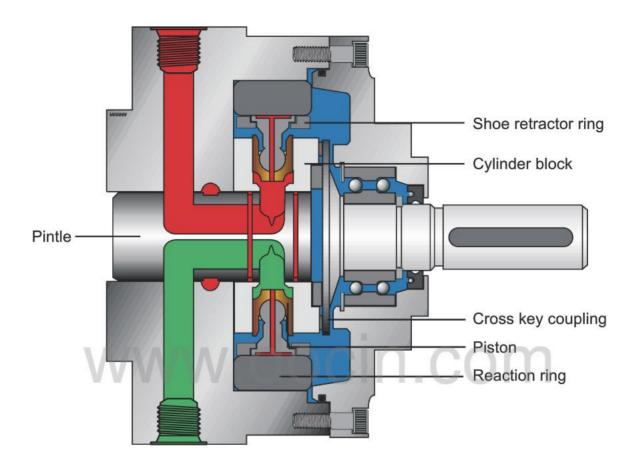


Figure 15-30 Pintle porting in a radial piston pump

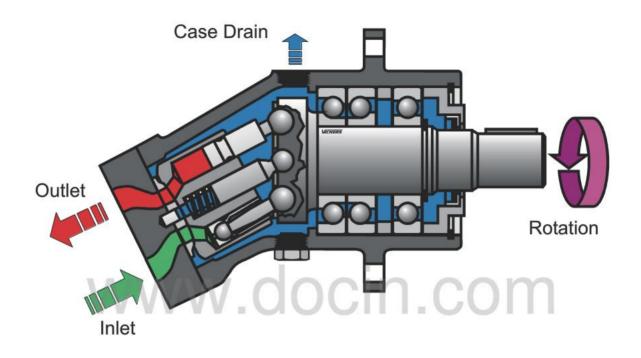


Figure 15-31 A bent-axis piston pump



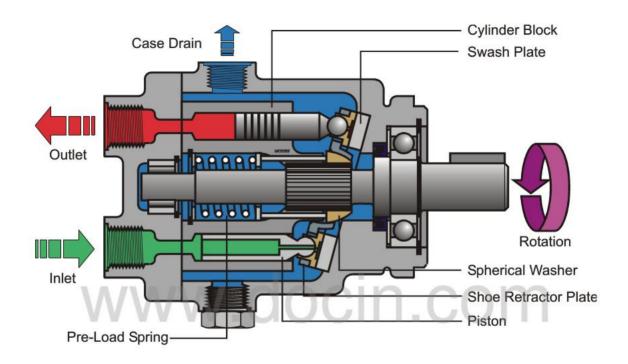


Figure 15-32 An in-line piston pump



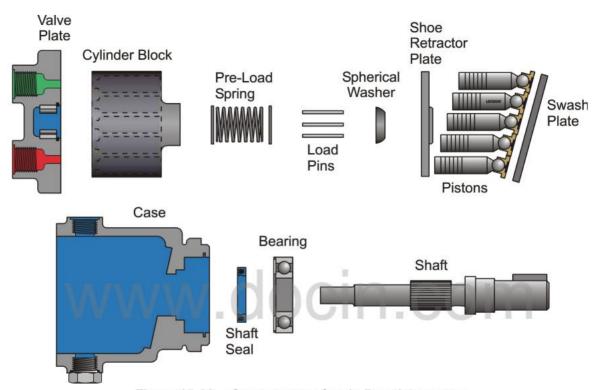


Figure 15-33 Components of an in-line piston pump

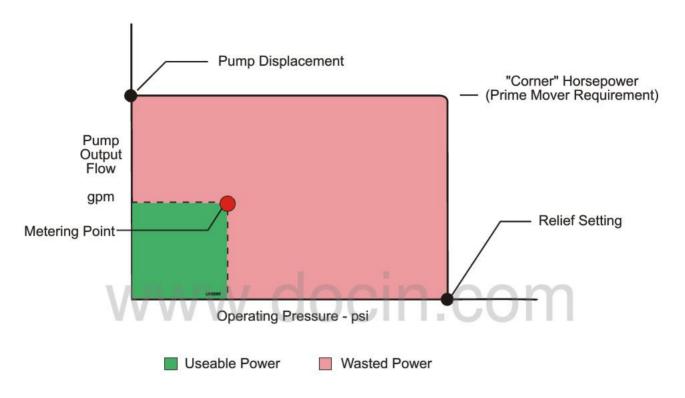


Figure 15-34 Metering losses in a fixed displacement system

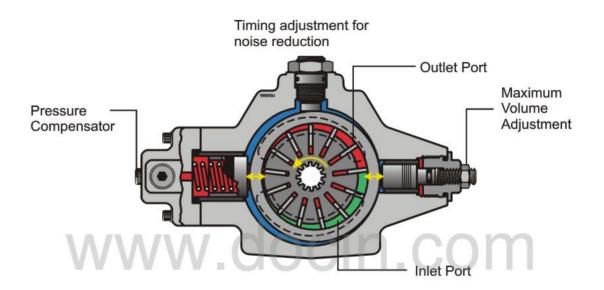


Figure 15-35 Variable Displacement Vane Pump



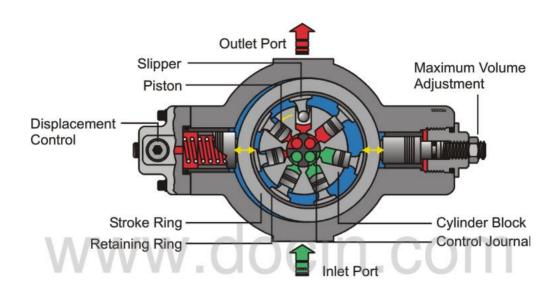


Figure 15-36 Variable displacement radial piston pump COPYRIGHT © (2001) EATON CORPORATION

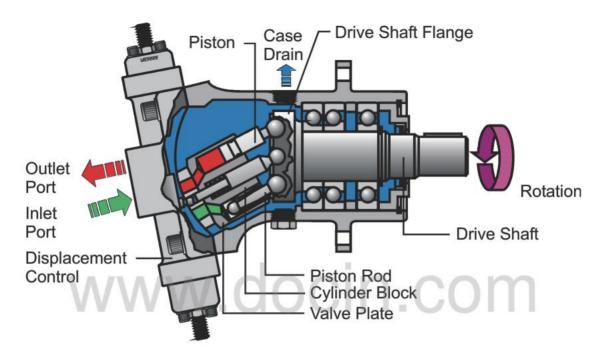


Figure 15-37 Variable displacement bent axis piston pump COPYRIGHT © (2001) EATON CORPORATION



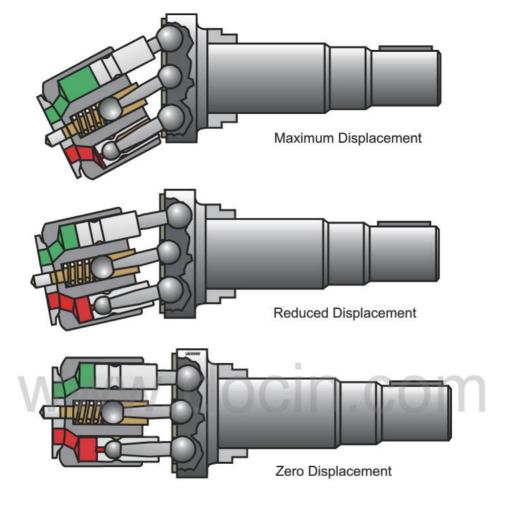


Figure 15-38 Displacement is controlled by the angle of the cylinder block

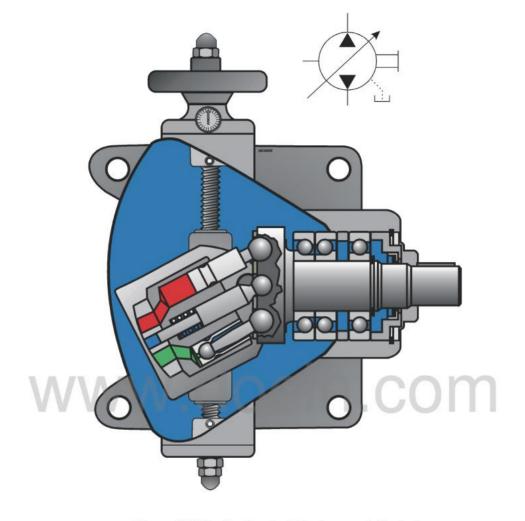


Figure 15-39 Mechanical Displacement Control

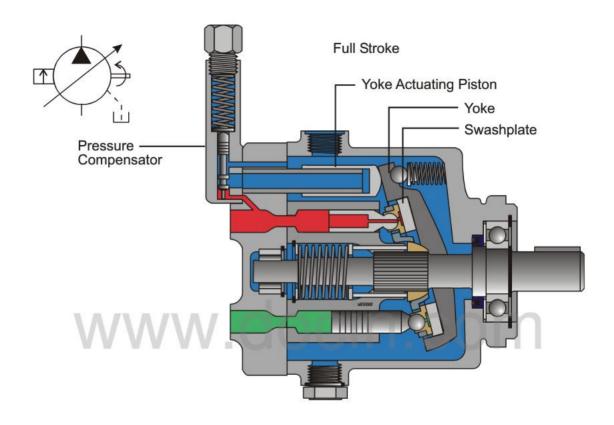


Figure 15-40 Cross section of an in-line piston pump



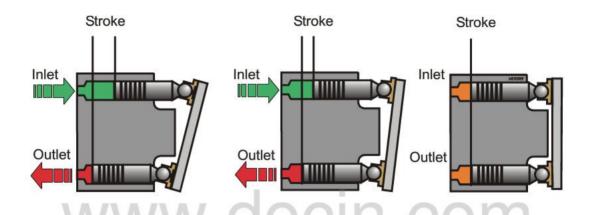


Figure 15-41 Pump displacement is based on swash plate angle COPYRIGHT © (2001) EATON CORPORATION

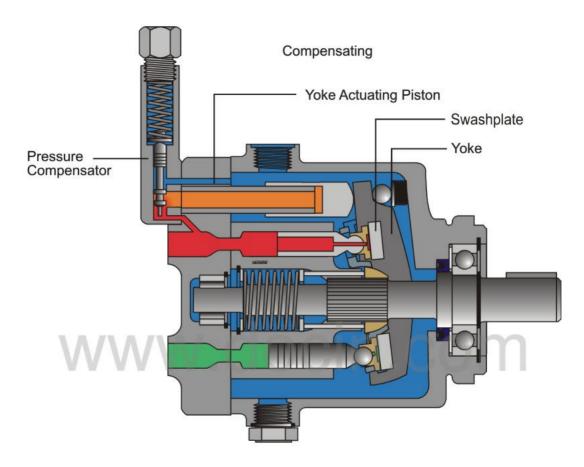


Figure 15-42 Compensators control the stroke adjusting piston



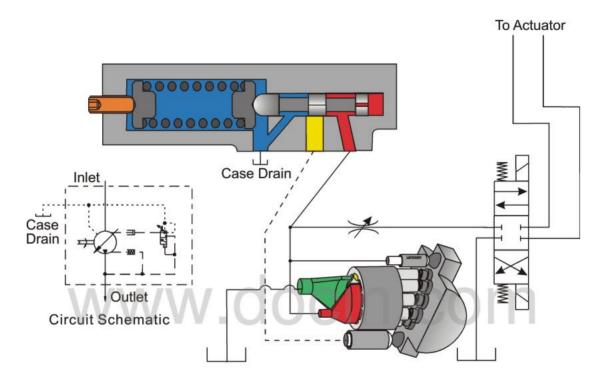


Figure 15-43 Pressure limiting compensator operation



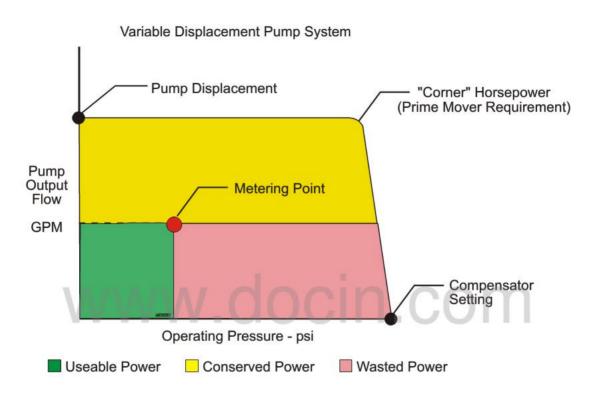


Figure 15-44 Energy saved by pressure compensation



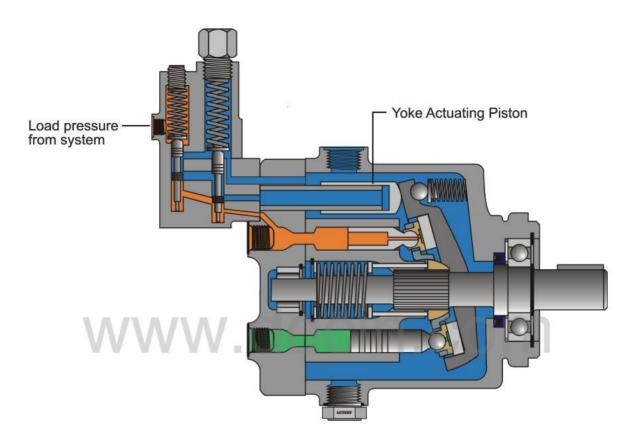


Figure 15-45 Piston pump with load sensing and pressure limiting compensator COPYRIGHT © (2001) EATON CORPORATION



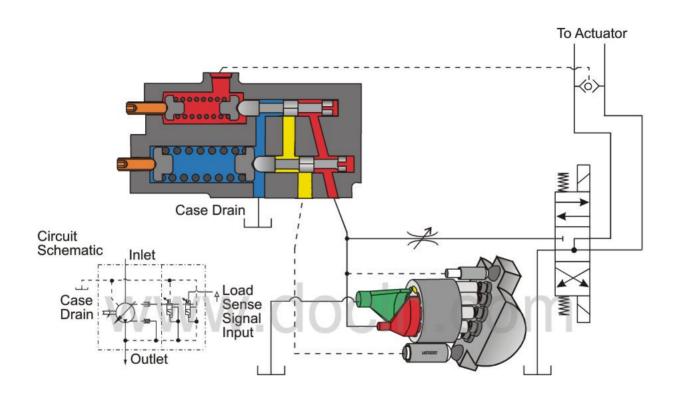


Figure 15-46 Load sensing / pressure limiting compensator operation COPYRIGHT © (2001) EATON CORPORATION

Variable Displacement Pump System

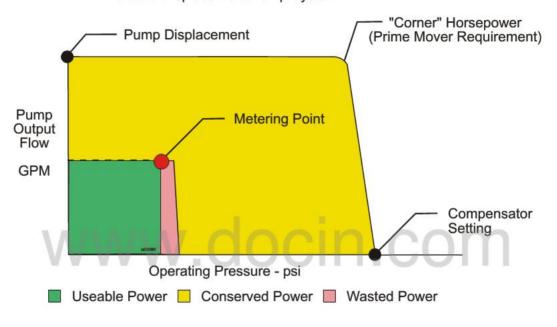


Figure 15-47 Energy savings caused by load sensing



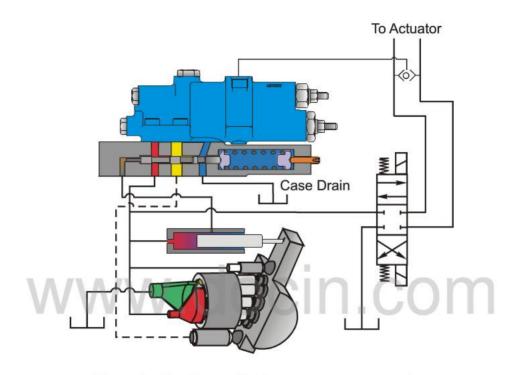


Figure 15-48 Torque limiting compensator operation COPYRIGHT © (2001) EATON CORPORATION

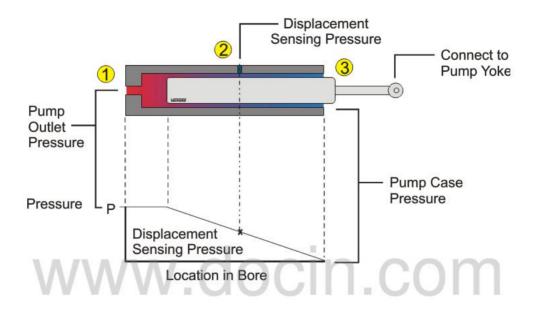


Figure 15-49 Yoke position sensing piston retracted



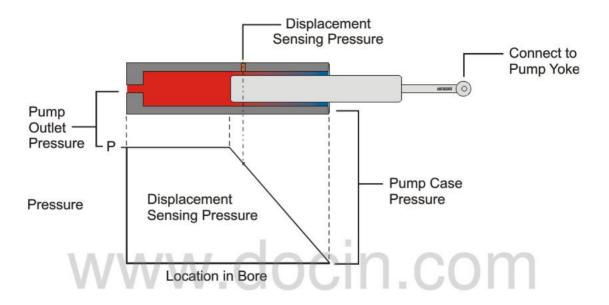


Figure 15-50 Yoke Position sensing piston extended



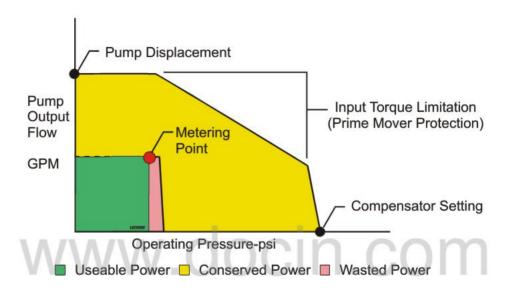


Figure 15-51 Reduced horsepower requirements by reducing corner horsepower COPYRIGHT © (2001) EATON CORPORATION



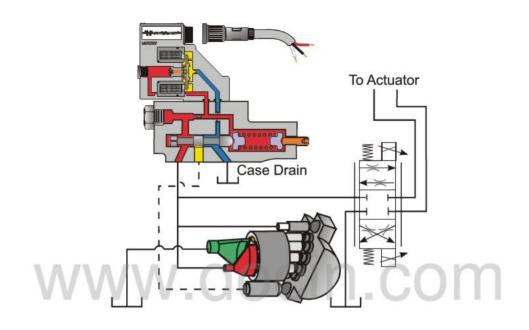


Figure 15-52 Electrohydraulic displacement contro COPYRIGHT © (2001) EATON CORPORATION